Sensitivity Analysis in Shape Optimization using Voxel Density Penalization

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Abstract

Shape optimization in the context of technical design is the process by which mechanical demands (e.g. loads, stresses) govern a sequence of piece instances, which satisfy the demands, while at the same time evolving towards more attractive geometric features (e.g. lighter, cheaper, etc.). The SIMP (Solid Isotropic Material with Penalization) strategy seeks a redistribution of local densities of a part in order to stand stress / strain demands. Neighborhoods (e.g. voxels) whose density drifts to lower values are considered superfluous and removed, leading to an optimization of the part shape. This manuscript presents a study on how the parameters governing the voxel pruning affect the convergence speed and performance of the attained shape. A stronger penalization factor establishes the criteria by which thin voxels are considered void. In addition, the the filter discourages punctured, chessboard pattern regions. The SIMP algorithm produces a forecasted density map on the whole piece voxels. A post-processing is applied to effectively eliminate voxels with low density, to obtain the effective shape. In the literature, mechanical performance finite element analyses are conducted on the full voxel set with diluted densities by linearly weakening each voxel resistance according to its diluted density. Numerical tests show that this approach predicts a more favorable mechanical performance as compared with the one obtained with the shape which actually lacks the voxels with low density. This voxel density - based optimization is particularly conventent for additive manufacturing, as shown with the piece actually produced in this work. Future endeavors include different evolution processes, albeit based on variable density voxel sets, and mechanical tests conducted on the actual sample produced by additive manufacture.

CCS Concepts

Glossary

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•*Applied computing* \rightarrow *Computer-aided manufacturing;* •*Computing methodologies* \rightarrow *Modeling and simulation;*

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Term	Description	Units
FEA	Finite element analysis	Adimensional
η ∈	Fraction of mass to be re-	Adimensional
(0,1)	tained in the final design	
$p \ge 1$	Penalty factor aimed to polar-	Adimensional
	ize element relative densities	
	around 0 and 1	
$R \ge 0$	Filter radius used to discour-	Adimensional
	age chessboard voxel patterns	
M_0	Initial mass of the domain	g
M	Mass function of the domain	gg
с	Compliance function of the	μJ
	domain	

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1. Introduction

Shape Optimization usually includes the set up of physical demands (stress, abrasion, vibration, light, heat, temperature, etc.) on
the desired object and a domain evolution, (reduction, in most publications). Evolution takes place until some constraint domain is
satisfied, both in terms of remaining volume and of responses to
the demands.

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(a) Design domain and boundary conditions.



(b) Example of the application of voxel density algorithms for shape optimization.

Figure 1: Design domain and result of the application of shape optimization.

This paper uses the term shape optimization as encompassing 10 11 both geometry and topology aspects. The reason for this usage is 12 that when voxel densities in one region vanish (geometry change), a side effect may be the creation of holes or disjoint portions, which 13 are topological changes. Therefore, topological changes derive in 14 natural form from geometry changes. Fig. 1 shows an example of 15 the application of shape optimization. 16

The strategy SIMP (Solid Isotropic Microstructure with Penal-17 ization [Sig01, LT14]) implies setting up of a goal percentage of 18 domain volume reduction, the decomposition of the domain in fi-19 nite elements, the load and boundary conditions. For the purpose 20 of the present discussion one may assume that the finite elements 21 are voxels. In each iteration of the algorithm, the density of each 22 voxel is re-considered to minimize the compliance of the piece, al-23 ways keeping the piece mass (i.e. summation of density times voxel 24 volume) below a certain level. 25

The voxel density strategy uses a parameter p to polarize the 26 densities of the finite elements towards 0 and 1. It also uses a fil-27 ter (parameter R) which discourages puncturing or chessboard ef-28 fects that would produce low and high density voxels mixing in a 29 non-dense pattern. The goal is, therefore, to have voxel - density -30 homogeneous regions. 31

This paper studies the influence of the parameters of the density-32 based algorithm, which is one of the most used structural optimiza-33 tion algorithms in additive manufacturing. For this purpose, a case 34 study in the field of solid mechanics is defined. This paper evalu-35 ates the impact of the density-based algorithm parameters, not only 36 in the geometry of the final design, but also in the structural perfor-37 mance and computation time. 38

The rest of the paper is organized as follows: Section 2 provides a review of the related literature. Section 3 describes the methodology used for testing the influence of the studied parameters. Section 4 presents and discusses the obtained results. Finally, Section 5 concludes the work and proposes some potential lines for future research.

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2. Literature Review 45

Section 2.1 shows the development of visualization tools to as-46 sist manufacturing processes. Subsequently, Section 2.2 presents 47 the use of structural optimization for additive manufacturing. Sec-48 tion 2.3 introduces the studies on the effects of the optimization 49 parameters in the solution given by the voxel density algorithm. Fi-50 nally, Section 2.4 concludes the literature review and synthesize the 51 contributions of this work. 52

2.1. Structural Optimization and Visual Computing for 53 **Assisting Manufacturing** 54

Structural optimization may be traced back to the work in [Ben89] 55 and has evolved rapidly since the beginning of the 20's. Applications in aerospace [SB11], fluid flow [KPEM10] and biomedicine [SPMN10] show the adoption of structural optimization in different fields. The reader is referred to the works in [DG14, SM13] for a more detailed review. Section 2.2 discusses the use of structural optimization in manufacturing.

In recent years, different tools of visual computing have started to support structural optimization and manufacturing [MHSL18, MMA*14, WWZW16], proving that visual computing is a core technology of Industry 4.0. [PTB*15]. This paper states the mathematical and algorithmic background for the development of an interactive and intuitive tool to assist the process of structural optimization in additive manufacturing.

Visual computing is a core technology in Industry 4.0. this realm, interactive and intuitive graphical environments have essential role in the assistance of modern manufacturing processes [PTB*15, BA13]. Visual computing impacts both the design [DHL14] and the manufacturing process [MMA*14].

recent years, different techniques of computational have been applied in additive manufacturing. geometry Important necessities are the design of efficient support structures [VGB14] and materials with specific geometrical and mechanical characteristics [MHSL18, WWZW16]. The simulation of the printing process also pose challenges in computer graphics, such as the generation of effective and efficient meshing algorithms [LCA18].

2.2. Structural Optimization for Additive Manufacturing

Although structural and shape optimization impact diverse manufacturing methods, additive manufacturing is particularly convenient for materializing voxel scale optimization. In the context of additive manufacturing, optimization is conducted by (a) growing / clipping the shape (i.e. bi-directional evolutionary structural optimization -BESO [TKZ15, TDZZ18, MZARS*19]), (b) tuning the density of spatial neighborhoods ([Lan16, PAHA18, ZCX19]), (c) using level sets to determine infill and shell profiles ([LYT18,FLGX19]), and (d) tuning diameters (proportional topology [CZB*17]).

Voxel density as tuning parameter has been used along level sets as supports for shape optimization in the context of additive manufacturing ([LM16]). Voxel density variations are relevant in various additive manufacturing aspects, such as: (1) minimization of

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support structures during the material deposition stages, (2) gener- 150 97

ation of lattice and porous structures for weight reduction, and (3) 151 98

tailoring part designs for additive manufacturing. 99

153 Ref. [Lan16] presents neighborhood density optimization 100 which hosts elimination of deposition stage support structures. 101 155 Ref. [PAHA18] maps density maps onto lattice materials suited for 102 shape optimization. Ref. [ZCX19] finds voxel density maps which 156 103 optimize shape, while at the same time integrates an overhang con-104 straint into the formulation of the shape optimization with additive 105 158 manufacturing. 106

2.3. Effect of the Parameters in Voxel-density Algorithms 107

As shown in the previous section, voxel density algorithms have 108 been used in structural optimization for different and varied appli-109 cations. However, it is not clear how the parameters associated to 110 the optimization process affect, not only the topology and geome-111 try of the final design, but also other relevant variables, such as the 112 convergence speed, objective function, and structural performance 113 of the obtained design. 114

The impact of the penalization factor p in the geometry of the 115 final design has been widely studied. It is known that large pe-116 nalization factors (p > 3) tend to produce black-and-white de-117 signs [Sig01, LT14, DHV09, AAH* 10, GWH17, VBSDC18]. How- 169 118 ever, the influence of the penalization factor on the behavior of 170 119 other variables (e.g. compliance and von Mises stress) has not been 120 established. 121

On the other hand, it is common to use filtering techniques 122 to reduce the checkerboard patterns that result from numerical 123 instabilities of the density-based methods [BS04]. In this case, 124 174 a filtering radius R must be included. This parameter defines 125 the area of the neighborhood in which the filter is applied. The 126 larger the filtering radius R, the simpler the geometry of the final 127 128 shape [GAV16, GWH17]. However, the impact of this parameter on 177 129 the compliance, time of convergence and structural performance is not well studied yet. 130

Ref. [GAV16] studies the effects of the variation of the goal vol-131 ume/mass fraction in the geometrical complexity of the obtained 132 designs. Refs. [EKB07, AAH*10] state that different designs can be 133 obtained by varying the initial density distribution. Besides, other 134 parameters concerning the finite element analysis (FEA) are also 135 studied. Ref. [DHV09] shows the advantage of quadratic finite ele-136 ments over the linear ones for avoiding checker-board patterns and 137 Ref. [EKB07] exhibits the mesh density dependency of the geom-138 etry of the final solution. However, these analyses mainly focus on 139 the geometry of the final shape, leaving aside the structural and 140 mechanical performance of the piece. 141

2.4. Conclusions of the Literature Review 142

The interest of the additive manufacturing community to advance 143 towards structurally optimal designs has been shown. Different 144 structural optimization algorithms (e.g. density-based, level set, 145 evolutionary structural optimization) have been used in the context 186 46 of additive manufacturing. However, the success of the optimiza-47 187 tion is highly dependent on the chosen parameters associated to the 148 188 algorithm. 149 189

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This paper focuses on getting a better understanding on how the parameters of the voxel density method affect (1) the behavior of the algorithm and, (2) the geometry and structural performance of the obtained design. This literature review has shown that exist few works that tackle this task. Most of the studies limit to evaluate only changes on the final geometry.

This work assess (1) the speed of convergence of the algorithm, (2) the final compliance, (3) the final maximum von Mises stress and, (4) the geometry and manufacturability of the final shape. As opposed to the found in previous works-in which the tested design is the voxel density map-measurements are also taken on the final piece.

3. Methodology

3.1. Tuning of Element Density

The objective of the classical structural optimization algorithms is to minimize the amount of material of a design so that it remains functional. In particular, density-based methods for shape optimization aim to find the optimal distribution of the relative densities (x_i) of the FEA elements along the domain.

In order to avoid FEA elements with intermediate (gray) densities-i.e. densities that are neither close to 0 nor 1-, voxel density methods adopt the rule in Eq. 1:

$$E_i = x_i^p E_0 \tag{1}$$

where p is the penalization power for intermediate densities and, E_i and E_0 are the elastic moduli of the *i*-th element and the raw material, respectively.

The formulation for the minimization of compliance in Eq. 2 ([Sig01,LT14]) assumes that the domain is (1) rectangular prismatic, and (2) discretized into N cubic FEA elements (voxels):

minimize
$$c(\mathbf{X}) = \mathbf{U}^{T}\mathbf{K}\mathbf{U}$$

subject to $M(\mathbf{X}) \le \eta M_{0}$, (2)
 $\mathbf{K}\mathbf{U} = \mathbf{F}$,
 $0 \le x_{\min} \le x_{i} \le 1, i = 1, \dots, N$.

where $\mathbf{X} = [x_1, \dots, x_N]^T$ is the vector of relative densities, x_{\min} is the minimum value that the relative density can reach (non-zero to avoid discontinuities that can produce numerical issues), $c(\mathbf{X})$ is the compliance function, U is the global displacement vector, **F** is the global force vector, **K** is the global stiffness matrix, M_0 is the mass of the initial design domain, η is the fraction of mass that aims to be retained in the final design and $M(\mathbf{X})$ is the mass function (Eq. 3),

$$M(\mathbf{X}) = \frac{M_0}{N} \sum_{i=1}^{N} x_i.$$
 (3)

Most of the implementations of the voxel density algorithms also include filtering techniques to avoid checkerboard patterns and, mesh-dependent solutions [Sig01]. One of the most frequently used filters is the sensitivity filter, which operates on the derivatives of

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the compliance function, as shown in Eq. 4 [Sig01]: 190

$$\widetilde{\frac{\partial c}{\partial x_i}} = \frac{\sum_{j \in N_i} H_{ij} \frac{\partial c}{\partial x_j} x_j}{x_j \sum_{i \in N_i} H_{ij}}, \qquad (4)^{221}$$

where $N_i = \{j : \text{dist}(i, j) \le R\}$ is the neighborhood of the *i*-th ele-191 ment and R is the filter radius and, H_{ii} is a weight factor defined in 192 Eq. 5: 193

$$H_{ij} = R - \operatorname{dist}(i, j), \tag{5}$$

224 where dist(i, j) is the distance between the centers of the elements 194 *i* and *j* (c_i and c_j , respectively), divided by the length *l* of the FEA ²²⁵ 195 elements (Eq. 6): 226 196

dist
$$(i, j) = \frac{||c_i - c_j||}{l}$$
. (6) 228

3.2. Conversion of the Voxel Density Map to the 197 **Design-for-Manufacturing** 198

The output of the implemented algorithm is a density map 232 199 (Fig. 2(a)) in which each voxel *i* has an associated relative den- 233 200 sity x_i ($0 \le x_i \le 1$). In general, this design cannot be manufactured. ²³⁴ 201 In order to select the elements to manufacture, this paper employes 235 202 the algorithm presented in Ref. [SM13]. The algorithm finds the 203 minimum density threshold x_T that guarantees the mass constraint 204 for the design-for-manufacturing (also called black-and-white de-205 sign). The surviving elements are those for which $x_i \ge x_T$. Fig. 2 206 shows an example of the conversion of the voxeld density map to 207 the design-to-manufacturing. 208



Figure 2: Conversion of the voxel density map to the design-to-242 manufacture.

3.3. Sensitivity Analysis 209

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The formula in Eq. 7 allows the numerical analysis of the sensitivity 246 210 of the function F with respect to the parameter α : 247 211

$$= \frac{\partial \ln F}{\partial \ln \alpha} = \frac{\alpha}{F} \frac{\partial F}{\partial \alpha} \approx \frac{\overline{\alpha}}{\overline{F}} \frac{\Delta F}{\Delta \alpha}, \qquad (7) 249$$
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251 where $\Delta \alpha$ and ΔF denote small changes in the value of α and F; 212 252 and $\overline{\alpha} = \alpha + \Delta \alpha/2$, $\overline{F} = (F_{\alpha} + F_{\alpha+\Delta\alpha})/2$. 213 253

In this paper, the functions F to analyze are: compliance, maxi- $_{254}$ 214 mum von Mises stress and number of iterations. Likewise, the pa- 255 215 rameters α to study are *p* and *R*. 216 256

217 Relative sensitivity allows to study how slight variations in the 257 value of the parameters can affect the mechanical performance of 258 the final piece. 259

Von Mises stress is used in solid mechanics as a failure criterion and it is desirable to minimize it. Von Mises stress is defined as per Eq. 8:

$$\sigma_{VM} = \sqrt{rac{\left(\sigma_{1} - \sigma_{2}
ight)^{2} + \left(\sigma_{2} - \sigma_{3}
ight)^{2} + \left(\sigma_{3} - \sigma_{1}
ight)^{2}}{2}},$$

where σ_1, σ_2 and σ_3 are the principal stresses.

3.4. Case Study

This paper uses a case study for the analysis of the effects of the algorithm parameters. This section describes: (1) the domain and material used for the simulations and, (2) the configuration of the numerical tests.

3.4.1. Domain of Analysis and Material Characterization

The studied domain is a 3D fixed beam with linearly distributed load applied in the center of the top face (see Fig. 1(a)). The beam has size 140.0mm x 20.0mm x 20.0mm and the magnitude of the total applied load is 1.1N. The material employed for the simulations is a PLA filament of a commercial brand. The properties of this material are presented in Table 1.

Table 1: Properties of the PLA filament used for the simulations.

Propert	у	Value
Young's	modulus	1230 MPa [<mark>BQ18</mark>]
Poisson'	s ratio	0.33 [FAL16]
Density		1.24 g/cm ³ [BQ18]

The domain in Fig. 1(a) is symmetric to the planes depicted in Figs. 3(a) and 3(b). Therefore, it can be simplified to the domain in Fig. 3(c). The equivalent load case is shown in Fig. 4.

In order to show the equivalence of the load cases presented in Figs. 1(a) and 4, a FEA simulation is executed for each domain, using F = 1.1N. Results of the simulations are presented in Fig. 5. Notice how the displacements of the two load cases are equivalent. This result allows to execute the simulations of the shape optimization algorithm on the simplified domain.

3.5. Set-up of Numerical Experiments

This paper conducts studies of the effects of p (density polarization) and R (region homogenization) parameters upon the piece geometry and mechanical performance, in the scenario of voxel density optimization methods. Table 2 presents the set of different simulations used for the study of each parameter. The measured variables for each simulation are: (1) compliance, (2) maximum von Mises stress and, (3) convergence speed, measured by the number of iterations. The authors implemented the voxel density optimization method in C++. The implementation uses the optimality criteria for updating the variables within the optimizer [Ben95].

To execute the FEA simulations, the domain in Fig. 4 is discretized into 1750 voxels (35x5x10). Subsequently, the FEA mesh is obtained by converting every voxel into a regular hexahedral (cubic) FEA element.

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Figure 3: *Simplification of the domain in Fig. 1(a).*



Figure 4: Design domain and boundary conditions. Simplified domain.



Figure 5: Comparison of the X and Z displacements for the original and simplified domain.

4. Results and Discussion

Sections 4.1 and 4.2 discuss the influence of the penalty factor p and the filter radius R in the manufacturability, compliance and maximum von Mises stress of the final design, so as the convergence speed of the algorithm. Measurements are executed on both the voxel density map and the black-and-white design. Subsequently, Section 4.3 presents a sensitivity analysis of the studied variables with respect to p and R. Finally, Section 4.4 shows some of the specimens generated using different parameter configurations.

4.1. Influence of the Penalty Factor in the Geometry, Manufacturability and Mechanical Performance of the Design

To evaluate the influence of the penalty factor p in the geometry and structural performance of the final design, 14 simulations were executed varying the value of p between 1.0 and 7.5, as shown in 275 Table 2. Figs. 6(a), 6(b) and 6(c) show the resultant density field 276 for p = 1.0 (no penalty), p = 3.0 and, p = 7.0. Histograms in 277 Figs. 6(d), 6(e) and 6(f) depict the frequency distribution of the 278 density values. Notice that for p = 1.0, density distribution is con-279 centrated in the interval (0.0, 0.2). On the other hand, for p = 3.0280 and p = 7.0, the largest bars are for $x_i = 0.0$ and $x_i = 1.0$. This den-281 sity distributions show the action of the penalty factor to eliminate 282 the intermediate densities. 283

Table 2: Values of the parameters used for the numerical simula-tions.

Analyzed	Parameter value				
parameter	р	R	η	M_0	
р	$\{1.0, 1.5, \dots, 7.5\}$	1.0	0.1	17.4 g	
R	3.0	$\{0, 1, \dots, 5\}$	0.1	17.4 g	

Figs. 6(g), 6(h) and 6(i) display the black-and-white design for p = 1.0, p = 3.0 and, p = 7.0. The design for p = 1.0 is composed by multiple non-connected parts and cannot be manufactured. The differences in the designs for p = 3.0 and p = 7.0 show that larger values of p tend to produce simpler geometries.

Fig. 7(a) shows the compliance of the gray and black-and-white designs of the 14 simulations varying *p*. For p = 1.0 and p = 1.5 the black-and-white domains are not connected and, therefore, compliance is not reported. Notice that for the gray domain, compliance tends to increase as *p* increases. However, for the black-and-white design, compliance converges to a value close to $4.0 \ \mu$ J.

Fig. 7(b) displays the maximum von Mises stress for the gray and black-and-white domains. So as in the case of compliance, maximum von Mises stress has a different behavior for the gray and black-and-white designs. In the case of the gray domain, maximum von Mises stress tends to increase, even for $p \ge 2.0$. On the other hand, for the black-and-white domain, maximum von Mises stress oscillates around 100 kPa.

Notice that for the gray domain is analyzed, For the studied gray domains, the compliance and maximum von Mises stress attain their lowest values when p = 1.0 and p = 1.5. However, for



Figure 6: Impact of the penalty factor in the geometry and manufacturability.



Figure 7: Impact of the penalty factor in the convergence speed and mechanical performance.

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these values of p, the respective black-and-white domains can- $_{321}$ 305 not be manufacture. It exhibits that the results for the black-and- 322 306 white domain are not necessarily in concordance with the results 307 for the gray domain. It demonstrates the importance of analyzing 323 308 the black-and-white domain, which is the one to be manufactured. 324 309 325 In Fig. 7(c) can be seen the number of iterations that the algo-310 326 rithm needed to converge for every value of p. The reader can see 311 327 that, for the domains that can be manufactured (p > 2.5), large val-312 328 ues of p tend to accelerate the convergence of the algorithm. 313 329 330

4.2. Influence of the Filter Radius in the Geometry, Manufacturability and Mechanical Performance of the Design

To study the influence of the filter radius *R*, it was varied between 334 0.0 and 6.0. Figs. 8(a), 8(b) and 8(c) show the resultant density field 335 for R = 0.0 (no filtering), R = 1.0 and, R = 3.0. Figs. 8(d), 8(e) 336 and 8(f) show the corresponding histograms of the density maps: 337 when R increases, the density is distributed more evenly along the domain and, therefore, more intermediate densities appear.

The black-and-white domains for R = 0.0, R = 1.0 and, R = 3.0 are displayed in Figs. 8(g), 8(h) and 8(i). Complex and detailed geometries are attained for small values of R. However, the geometrical complexity stimulates the appearance of non-manufacturable sub-domains. Fig. 12(c) show that for R = 0.0 appear voxels that are connected by a single edge, which impedes the correct manufacturing (even using additive manufacturing technologies) of the piece. The occurrence of these chessboard patterns are associated to numerical errors that may be caused by the voxel discretization and the type of FEA element used for the simulations [PQR05].

The compliance and maximum von Mises stress are shown in Figs. 9(a) and 9(b). For R = 5.0, compliance and maximum von Mises stress are not reported for the black-and-white domain because the domain is not connected. The increase of the compliance for the gray domain (Fig. 9(a)) for increments in R is noticeable.



Figure 8: Impact of the radius filter in the geometry and manufacturability.



Figure 9: Impact of the filter radius in the convergence speed and mechanical performance.

However, the value of R does not affect the compliance of the black- 355338 and-white domain. 356 339

a faster convergence. However, the final design may not be manufacturable. Therefore, intermediate values of R should be selected.

So as in the previous section, the behavior of the compliance and 340 maximum von Mises stress is different for the black-and-white and 341 gray domains. The mechanical performance of the gray domain is 357 342 merely illustrative and does not represent a real piece. Therefore, it 343 358 is necessary to check the performance of the piece for manufactur-344 359 ing. This finding shows the relevance of a stage of validation in the 345 360 pipeline of structural optimization. 346 361

Fig. 9(b) shows that larger values of R produce structures with 362 347 larger maximum von Mises stress for the black-and-white domain. 363 348 This result agrees with the result for the gray domain when $R \le 3.0$. 364 349 However, for $R \ge 4.0$, the maximum von Mises stress of the gray 350 365 domain decays. It is related to the more even distribution of the 351 366 relative densities in the volume. 367

Fig. 9(c) shows the convergence speed of the algorithm depend- 368 ing on the value of R. No filtering and large filter radii contribute to 369

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4.3. Sensitivity Analysis

Fig. 10 presents the relative sensitivity of the compliance, maximum von Mises stress and number of iterations with respect to the parameter p. To calculate these values, R was fixed to 1.0. It is noticable noticeable that for p > 4.0, the compliance and the maximum von Mises stress are not affected by the value of p. On the other hand, the convergence speed of the algorithm is very sensitive to the value of p.

Fig. 11 displays the sensitivity analysis of the parameter R for the studied variables: compliance, maximum von Mises stress and convergence speed. From Figs. 11(a) and 11(b) can be infered that R does not have much influence on the compliance and maximum von Mises stress of the final design. However, R does impact the me-



Figure 10: Relative sensitivity of the compliance, maximum von Mises stress and convergence speed w.r.t.



Figure 11: Relative sensitivity of the compliance, maximum von Mises stress and convergence speed w.r.t. R.

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chanical performance of the voxel density map, specially for larger 395

values of *R*. Convergence speed is also affected when $R \ge 2.0$.

4.4. Evaluation of the Manufacturability and 3D Printed Pieces

402 Additive manufacturing allows the production of complicated ge-374 403 ometries that cannot be manufactured using other technologies. To 375 404 evaluate the feasibility of the designs produced by the voxel den-376 405 sity algorithm, three resultant domains of Sections 4.1 and 4.2 were 377 selected. Figs. 12(a)-12(c) show the corresponding STL model of $_{406}$ 378 each design. The domain in Fig. 12(c) has neighborhoods in which $_{407}$ 379 the voxels are connected only by an edge, which compromises the $_{408}$ 380 manufacturability of the piece. 381 409

410 Figs. 12(d)-12(f) show the 3D printed pieces obtained from the 382 411 STL models in Figs. 12(a)-12(c). Notice that for the first two do-383 mains, the geometry of the shape can be reproduced accurately. 384 However, due to the single edge's connections in the third design, 385 some sub-domains disconnect when support material is removed. 386 412 It shows the importance of the filtering techniques for suppressing 387 punctured and chessboard pattern regions. In order to improve the 413 388 manufacturability of the final piece, different solutions for sup- 414 389 pressing these punctured or chessboard-looking regions have been 415 390 proposed. Filtering techniques (as the implemented in this work), 416 291 the use of higher-order FEA elements and the deletion of single- 417 392 edge or single vertex connections [PQR05] are some of the plausi- 418 393 ble solutions. 419

5. Conclusions

This paper presents analysis of the effects of the parameters of the voxel density algorithms in (1) the geometry and structural performance of the final design and, (2) the convergence speed of the algorithm. For the study, the authors use one set-up, therefore conclusions on the detailed behavior of the parameters may not be drawn. However, results show that (a) extreme values of the parameters can affect the manufacturability and mechanical performance of the designs and (b) mechanical analyses must be executed on the domain-to-manufacture and not in the *optimal* voxel density map given by the algorithm.

Shape optimization is an intermediate step in the work-flow of the design-to-manufacturing. In this realm, it is important to understand how the shape optimization algorithms work and how their parameters affect the obtained design. This work can be a worthy tool for many designers and engineers that use commercial software that implements density-based methods.

5.1. Limitations

This work studies the effects of the penalty power p and the filter radius R independently. It may be interesting to understand the interaction between these two parameters. Future research should address the analysis of simultaneous changes in the values of p and R. Moreover, other parameters (e.g. mass fraction η) can be investigated. Physical experimentation is also required for testing the correctness and exactitude of the numerical results.



Figure 12: 3D printed designs obtained using the voxel density algorithm.

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5.2. Future Work 420

The authors look forward to generate an interactive tool to assist the 421 design process in additive manufacturing. The tool would allow de- $\frac{1}{461}$ 422 signers to visualize different different pieces and their mechanical 462 423 performance. It has to be capable of generating different configu- 463 424 rations for the domain, loads, constraints and parameter configura- 464 425 tions for shape optimization. 426

It is necessary to validate the conclusions drawn in this work. In $_{
m 467}$ 427 that sense, there are three line that are open for future research: (1) $_{468}$ 428 the simulation of other domains with different load cases, (2) the 469 429 analysis of correlations between p and R and (3) physical tests to 470 430 confirm numerical results. 471 431

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